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(54) Rotor for a permanent-magnet motor.

(57) With a view to reduce vibrations generated by a motor and improve resistance to heat, reliability and vibration isolation properties without increasing the cost or the number of manufacturing steps in a rotor for a permanent-magnet motor, a rotor core is formed of an outer peripheral portion by which the permanent magnet is fixed, and a central portion by which a shaft is fixed, a clearance is provided between the outer peripheral portion and the central portion, and the outer peripheral portion is connected to the central portion by means of a plurality of fastening members. Thus, a rotor having excellent vibration isolation properties can be obtained. Since the rotor core is formed of laminated iron plates on which the outer peripheral portion, the central portion and the fastening members are all formed as one piece, resistance to heat, reliability and the vibration isolation properties can be enhanced without in-

creasing the cost or the number of manufacturing steps.

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BACKGROUND OF THE INVENTION

Field of the Invention:

The present invention relates to a rotor for a permanent-magnet motor having a vibration isolation construction for use in air conditioners, hot-water supply units or the like.

Description of the Related Art:

In recent years, permanent-magnet motors are becoming widely used in fan motors because of good controllability. A conventional rotor of a permanent-magnet motor will be described below with reference to Figs. 6, 7A and 7B.

Fig. 6 illustrates a DC brushless motor, one type of a permanent-magnet motor. As shown in Fig. 6, the permanent-magnet motor comprises a stator core 9 and a rotor core 6. A permanent magnet 5 is fixed around the outer periphery of the iron core 6, a shaft 7 is fixed around the central portion thereof, and the shaft 7 is rotatably assembled onto a bearing 8 of a rotor frame in which the rotor core 6 is disposed. A fan 10 is fixed to the end portion of the shaft 7 by means of a nut 13.

With the above-described construction, since the rotor core 6 has insufficient vibration attenuation properties, vibrations along the path of rotation generated in a rotor due to rotary magnetic fields of the stator are transmitted to the fan 10 via the shaft 7, causing the fan 10 to vibrate and make an unpleasant noise.

As a measure for preventing such unpleasant noise, there is a commonly used means whereby rubber 11 is assembled on both sides in a direction perpendicular to the thrust of the shaft 7 by a sandwich structure by making use of a fan receiving plate 14 for reducing noise generated due to resonance of the fan 10 when the fan 10 as shown in Fig. 8A is fixed to the rotary shaft 7, or means whereby rubber 12 is inserted between the shaft 7 and the fan 10 by means of bosses 15 and 16. Rubber is commonly used to prevent vibration. As a means for reducing generated noise, from an electrical point of view, skew magnetization is performed.

However, in the prior art, since use of rubber increases the number of assembly steps or make the shape of the fan complex, the cost is high, and resistance of rubber to heat is problematical. Also, when skew magnetization is performed, efficiency, output and the like decrease due to loss of magnetic flux.

SUMMARY OF THE INVENTION

The present invention solves the above-mentioned problems of the prior art. It is an object of the present invention to provide a rotor for a permanent-magnet motor having excellent resistance to heat, reliability, and vibration isolation properties.

To achieve the above object, in the rotor of the permanent-magnet motor in accordance with the present invention, a permanent magnet is fixed around the outer periphery of a rotor core, the central portion being fastened to a shaft, a clearance being provided between the outer periphery and the central portion thereof, and the outer periphery being connected to the central portion by means of a plurality of fastening members. The rotor core is so constructed that the outer periphery, the central portion and the fastening members are made in one piece.

With the above-described construction, the present invention makes it difficult for vibrations along the path of rotation to be transmitted to the shaft. The operation thereof will be described below.

Fig. 9 illustrates an example of a torsion vibration system. The natural frequency of a torsion system can be generally determined by equation (1) below:

$$\text{Natural frequency} = (K(J_1 + J_2)/J_1 J_2)^{0.5}/2\pi \text{ (Hz)} \quad (1)$$

J_1 : inertia of rotor

J_2 : inertia in which shaft, fan and the like are taken into consideration

K : spring constant of a screw along the path of the rotation of a rotor, generated as a result of the outer periphery of the rotor core being connected to the central portion by means of a plurality of fastening members.

The natural frequency of equation (1) should be $1/2^{0.5}$ or less of the cut-off frequency on the basis of vibration isolation theory. The larger the difference, the larger the vibration isolation effect. When it is so set, the rigidity of the fastening members may be decreased in terms of vibration isolation performance up to the point that the fastening members can still withstand the force for transmitting a required torque and up to the point that the shaking at the end portion of the shaft is maintained within an allowable range depending upon use. The clearance between the stator and the rotor which should be secured in the motor is uniformly maintained by means of fastening members having great rigidity radially.

The above and further objects and novel features of the invention will more fully appear from the following detailed description when the same is read in connection with the accompanying drawings. It is to be expressly understood, however, that the drawings are for the purpose of illustration only and are not intended as a definition of the limits of the invention.

BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1A is a plan view of a rotor for a permanent-magnet motor in accordance with a first embodiment of the present invention;
 Fig. 1B is a sectional view taken along the line A-A of Fig. 1A;
 Fig. 2A is an enlarged sectional view of the rotor of the permanent-magnet motor in accordance with the first embodiment of the present invention;
 Fig. 2B is a perspective view of a fastening member shown in Fig. 2A;
 Fig. 3 is a sectional view of a rotor in accordance with another embodiment of the present invention;
 Fig. 4 is a sectional view illustrating a rotor in accordance with still another embodiment of the present invention;
 Fig. 5 is a sectional view of a rotor in accordance with yet another embodiment of the present invention;
 Fig. 6 is a sectional view of a conventional permanent-magnet motor;
 Fig. 7A is a plan view of a conventional rotor for a permanent-magnet motor;
 Fig. 7B is a sectional view of the conventional rotor for the permanent-magnet motor shown in Fig. 7A;
 Fig. 8A is a sectional view of conventional vibration isolation means;
 Fig. 8B is a sectional view of another conventional vibration isolation means; and
 Fig. 9 is a perspective view illustrating an example of a torsion vibration system.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Preferred embodiments will be explained below with reference to Figs. 1A and 1B, and Figs. 2A and 2B. Components which are the same as those explained in the prior art are given the same reference numerals.

As shown in the figures, a rotor core has an outer peripheral portion 1 by which a permanent magnet 5 is fixed thereto, and a central portion 2 by which shaft is fixed thereto. The outer peripheral portion 1 is connected to the central portion 2 by

means of a fastening member 3, and a clearance 4 is formed. The rotor core is formed by laminating a plurality of iron plates having the outer peripheral portion 1, the central portion 2 and the fastening member 3, all formed as one piece.

Next, the operation of the rotor for a permanent-magnet motor constructed as described above will be explained with reference to Figs. 2A and 2B. Fig. 2A is an enlarged view of the fastening member 3. Fig. 2B is a perspective view of only the fastening member 3. Rigidity of the fastening member 3 can be selected on the basis of width W and length L.

Rigidity of the fastening member 3 is determined by a cross-section second-order moment due to the shape shown in Fig. 2B, $I = bW^3/12$ (b: thickness of the laminated iron cores), and Young's modulus E. Therefore, rigidity K' is generally expressed as $K' = EI$. When the length L of the fastening member 3 is taken into consideration for K', eventually a spring constant K becomes K in equation (1). By selecting K appropriately, the frequency can be easily made $1/2^{0.5}$ or less of the frequency which cuts off the natural frequency, making it possible to prevent vibrations due to cogging torque generated along the path of the rotation of the rotor.

When the rotor of this embodiment and the conventional rotor were operated by a DC brushless fan motor shown in Fig. 6 and compared, it was confirmed that noise generated by the DC brushless fan motor, though not shown, is reduced by approximately 10 dB from the noise level of the fan motor which used the rotor of this embodiment. The rotor core of this embodiment is formed of laminated iron plates on which the outer peripheral portion 1, the central portion 2 and the fastening member 3 which connects the two portions, are all formed as one piece. This embodiment can be realized without specially providing fastening members and without increasing the cost or the number of manufacturing steps.

Figs. 3 and 4 illustrate another embodiment in which the shape of the fastening member which connects the outer peripheral portion 1 to the central portion 2 is varied. Fig. 3 illustrates a bent fastening member 17, and Fig. 4 illustrates a scrolled fastening member 18. There is another embodiment of a rotor core, shown in Figs. 1A, 3 and 4, formed by laminating iron plates, in which embodiment the iron plates are shifted one by one or in units of several plates along the path of the rotation and laminated. Fig. 5 illustrates an embodiment in which after the clearance 4 is made as small as possible, an effective length L and width W are secured for a fastening member 19. The same effect as the above can be obtained.

As can be understood from the above explanation of the embodiments, since in the present invention a permanent magnet is fixed onto the outer peripheral portion of a rotor core and coupled to a shaft at the central portion, a clearance is provided between the outer peripheral portion and the central portion, the outer peripheral portion and the central portion are connected by a plurality of fastening members, and the shape thereof is selected appropriately, vibrations can be effectively absorbed, and factors causing instability, such as deformation due to heat or aging, can be eliminated. No loss of magnetic fluxes due to skew magnetization occurs, achieving high efficiency and high output. In addition, since a rotor core is formed by punching iron plates by a press and laminating them, a rotor of a permanent-magnet motor having excellent durability, reliability and vibration isolation properties can be provided without increasing the cost or the number of manufacturing steps.

Many different embodiments of the present invention may be constructed without departing from the spirit and scope of the present invention. It should be understood that the present invention is not limited to the specific embodiments described in this specification. To the contrary, the present invention is intended to cover various modifications and equivalent arrangements included with the spirit and scope of the claims. The following claims are to be accorded the broadest interpretation, so as to encompass all such modifications and equivalent structures and functions.

Claims

1. A rotor for a permanent-magnet motor having a permanent magnet fixed onto an outer peripheral portion, having a shaft fastened onto a central portion, a clearance being provided between the outer peripheral portion and the central portion, and the outer peripheral portion and the central portion being connected by a plurality of fastening members.
2. A rotor for a permanent-magnet motor according to claim 1, wherein the outer peripheral portion by which a permanent magnet is fixed to a rotor core, the central portion by which the shaft is fastened, and the plurality of fastening members for connecting the outer peripheral portion to the central portion are all formed as one piece.

FIG. 1A

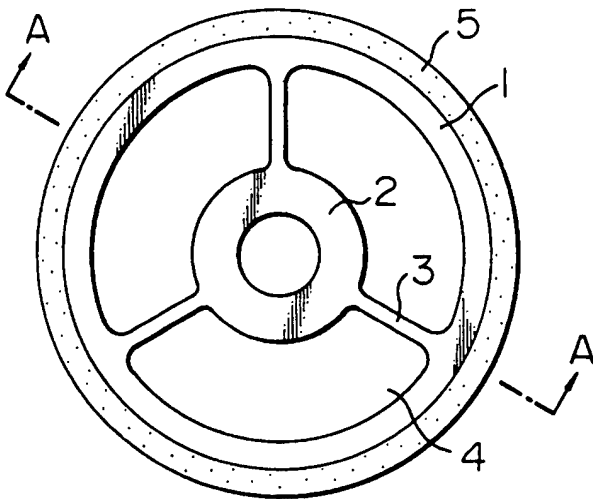


FIG. 1B



FIG. 2A

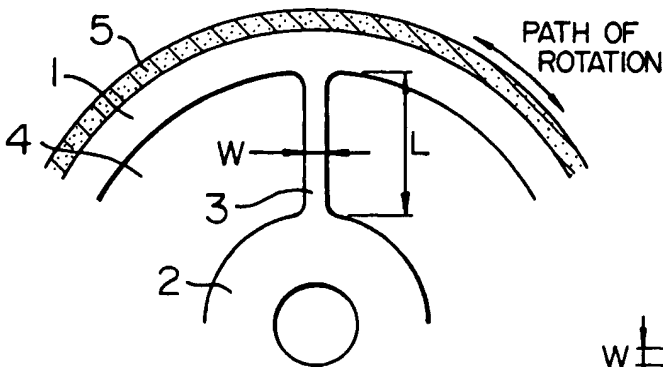


FIG. 2B

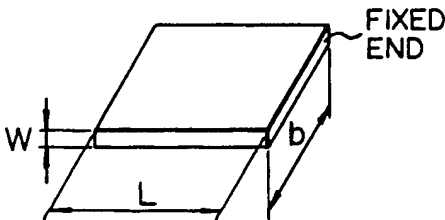


FIG. 3

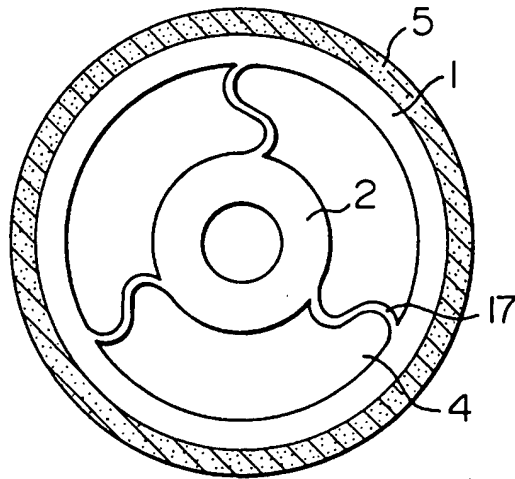


FIG. 4

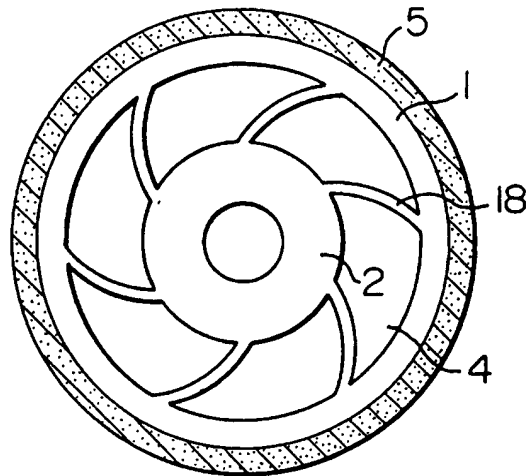


FIG. 5

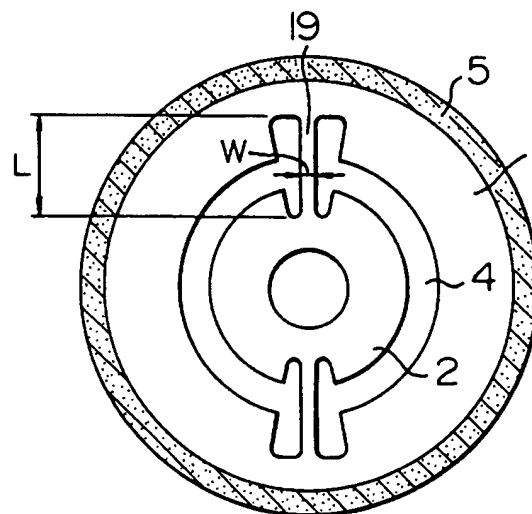


FIG. 6 PRIOR ART

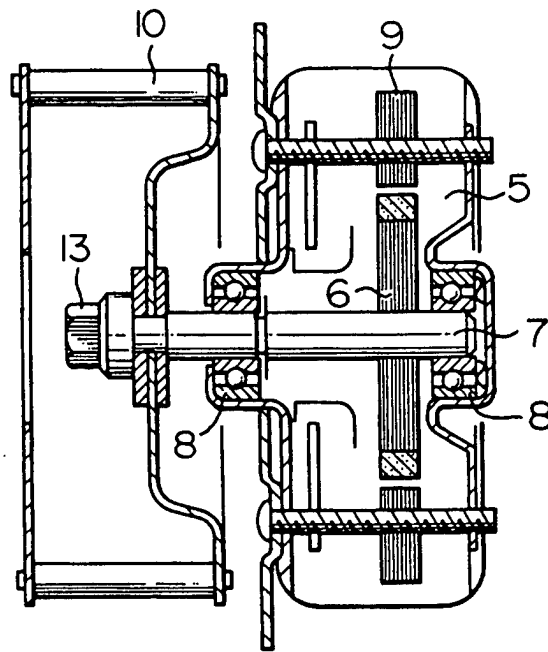


FIG. 7A PRIOR ART

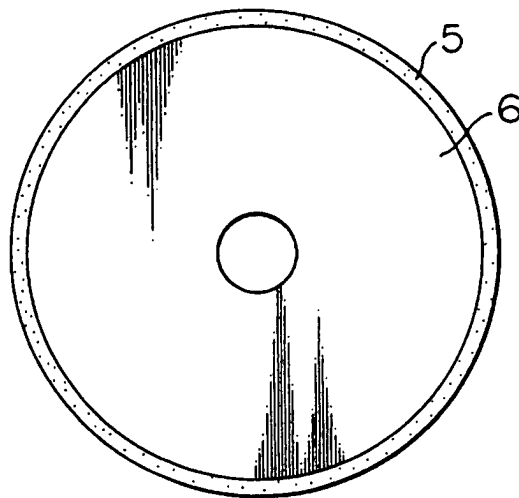


FIG. 7B PRIOR ART



FIG. 8A
PRIOR ART

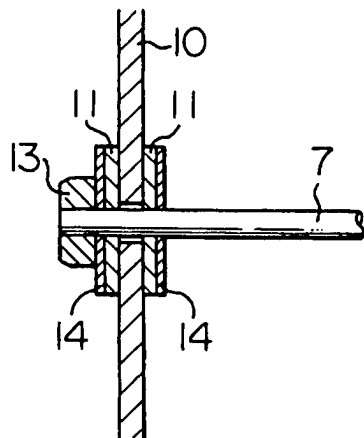


FIG. 8B
PRIOR ART

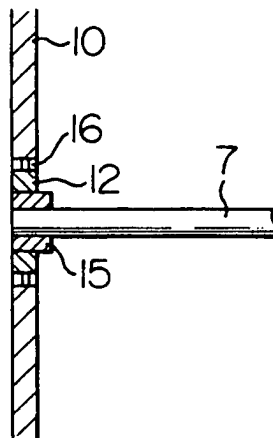
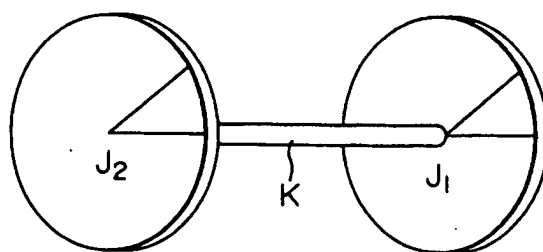


FIG. 9



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(54) **Rotor for a permanent-magnet motor.**

(57) With a view to reduce vibrations generated by a motor and improve resistance to heat, reliability and vibration isolation properties without increasing the cost or the number of manufacturing steps in a rotor for a permanent-magnet motor, a rotor core is formed of an outer peripheral portion (1) by which the permanent magnet (5) is fixed, and a central portion (2) by which a shaft is fixed, a clearance is provided between the outer peripheral portion and the central portion, and the outer peripheral portion

is connected to the central portion by means of a plurality of fastening members (17,18,19). Thus, a rotor having excellent vibration isolation properties can be obtained. Since the rotor core is formed of laminated iron plates on which the outer peripheral portion, the central portion and the fastening members are all formed as one piece, resistance to heat, reliability and the vibration isolation properties can be enhanced without increasing the cost or the number of manufacturing steps.

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FIG. 3

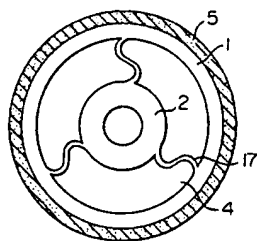


FIG. 4

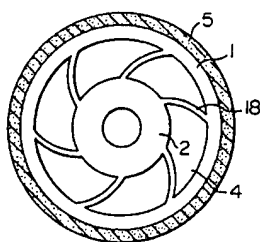
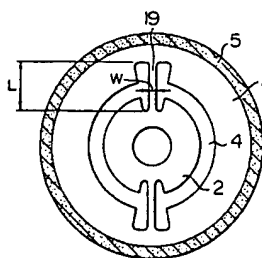


FIG. 5





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PAGE1

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X	SIEMENS POWER ENGINEERING & AUTOMATION vol. 8, no. 3, May 1986, DE BERLIN pages 147 - 149 H.GIERSE 'SIMODRIVE: Trendsetting Drive Systems for Machine Tools and Robots' * figure 4 * ---	1,2	
X	US-A-4 642 502 (K.H.CARPENTER & AL.) * column 2, line 31 - line 44; figure 1 * ---	1,2	
X	EP-A-0 413 183 (M.VENTURINI) * column 3, line 22 - line 30; figure 1 * ---	1,2	
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The present search report has been drawn up for all claims			
Place of search BERLIN		Date of completion of the search 25 AUGUST 1993	Examiner LEOUFFRE M.
CATEGORY OF CITED DOCUMENTS			
X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document		T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons * : member of the same patent family, corresponding document	



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DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.5)
A	US-A-4 472 650 (N.P.ADVOLTKIN & AL.) * column 4, line 65 - column 5, line 15; figures 5,6 * -----	1,2	
			TECHNICAL FIELDS SEARCHED (Int. Cl.5)
The present search report has been drawn up for all claims			
Place of search BERLIN		Date of completion of the search 25 AUGUST 1993	Examiner LEOUFFRE M.
CATEGORY OF CITED DOCUMENTS X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons A : member of the same patent family, corresponding document			

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